

1 **Archimedes screw generator powerplant assessment and field measurement campaign**
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3
4 **Abstract**

5 The Archimedes screw generator (ASG) is a cost-effective, eco-friendly micro-hydropower
6 technology that can operate effectively in low head and moderate flow rate environments. The
7 technology is well suited for distributed micro-hydropower and off-grid energy schemes, and its simple,
8 robust design allows for local manufacturing and installation in developing regions. There are many
9 successful ASG powerplants installed on- and off-grid, particularly in Europe. Archimedes screw
10 generators are largely designed by experience. As such, there is a need for an accurate performance
11 model in the literature to optimize plant design. There is a lack of experimental data in the literature,
12 particularly for large-scale, real-world screw plants. This study sought to gather both qualitative and
13 quantitative data and introduce it to the literature. The authors surveyed and measured various
14 Archimedes screw powerplants in the United Kingdom. The measurements were then added to a larger
15 dataset to validate a CFD simulation across a range of screw sizes: from a small 15 cm diameter
16 laboratory screw to a large 3.5 m diameter powerplant. The authors plan to use the CFD simulation's
17 accurate approximations to further the literature's ASG performance data for a wide range of screw
18 sizes, shapes, and operational parameter.

19 **Keywords**

20 Archimedes screw generator, Reverse Archimedes screw, hydrodynamic screw, micro-hydropower, off-
21 grid electrification, run-of-river, diversion hydropower

22 **Notation**

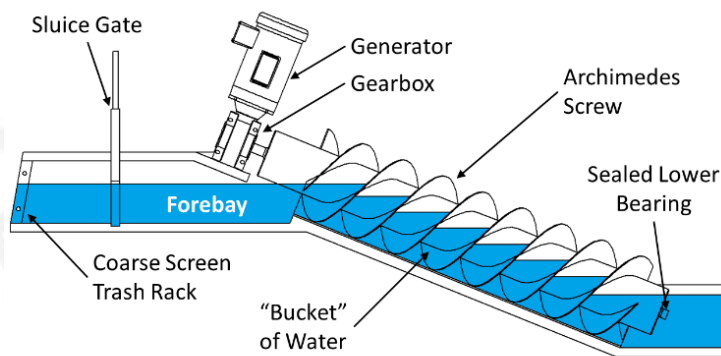
23 *The following symbols are used in this paper:*

- D_i = inner diameter of screw (m);
- D_o = outer diameter of screw (m);
- g = gravitational constant (m/s^2)
- G_w = blade-trough gap width (m)
- H = head (m);
- h_L = lower water level (m);
- h_U = upper water level (m);
- L = flighted length of screw (m);
- N = number of blades of screw (-);
- P = rated power (W);
- P_e = electrical power produced (W);
mechanical power converted by the screw
- P_s = (W);
- Q = flow rate (m^3/s);
- S = screw pitch (m);
- T = mechanical torque at the screw shaft (W);
- z_{wl} = bucket water level (m);
- β = inclination angle of screw ($^\circ$);
- η = mechanical efficiency of screw (-)
- ρ = density of working fluid (i.e. water) (kg/m^3)
- ψ_L = lower submergence level (-)
- Ω = rotation speed (rev/min);

24 **1. Introduction**

25 Archimedes screws are an ancient pumping technology that have also been used to generate
26 hydropower since the 1990s [1]. An Archimedes screw is made up of a helical array of simple blades
27 wrapped around a central, cylindrical tube – like a wood screw. In a power generating mode, the screw

28 rotates in the reverse direction that would be used for pumping, and hydro-electric energy is generated
29 mostly due to the static pressure differences on the screw blades (or “flights”) (Fig. 1); it is common to
30 refer to the technology as an Archimedes screw generator (ASG), or less commonly as a reverse
31 Archimedes screw (RAS) during this type of operation. Due to their simple, robust design ASGs are an
32 eco-friendly, low cost hydropower technology [2–4], well suited for installation in developing regions.



33
34 **Figure 1: Diagram of typical Archimedes screw generator installation.**

35 Rural, developing regions have little access to electricity. For example, only about 45% of the rural
36 population of Sub-Saharan Africa had access to electricity in 2018 [5]; an increase from 29% in 2009, but
37 still low [6]. Access to electricity is a priority for rural Cameroonians [7] as it provides increased health,
38 security, and productivity. It also gives better access to information, education, and entertainment [8].
39 Electricity reduces reliance on fossil fuels like diesel and kerosene by eliminating the need for kerosene-
40 based lanterns and wood-fueled cookstoves. An estimated 3 billion people worldwide cook using wood
41 biomass cookstoves or open fires; close to 4 million people die per year due to illness attributable to
42 indoor air pollution from these methods [9].

43 There was an estimated 30.4 GW of installed hydropower capacity in Sub-Saharan Africa in 2017
44 [10], however, 92% of an estimated 300 GW of potential is untapped [11]. Local availability and capital
45 expenses tend to be major hurdles in developing hydropower installations in rural developing regions
46 [7]. Equipment must either imported or manufactured locally. Imported systems tend to be more

47 sophisticated (by way of experience), but very costly to import and install. It can be very difficult for
48 local technicians to maintain complex imported systems, with access to parts being a particular issue.
49 Ho-yan et al. (2014) noted non-operational hydropower systems in Cameroon that could not be repaired
50 locally due to lack of experience or inability to access replacement parts [7]. The local manufacturing of
51 equipment and implementation of less complex technologies, such as Archimedes screw generator
52 which do not require precision machining, would mitigate many of these issues. Further, many ASGs
53 have been successfully retrofit to existing mill sites or flood control structures to reduce cost of civil
54 works [12].

55 The majority of active ASG powerplants are in the developed world, predominantly in Europe [13].
56 However, some ASGs have been successfully implemented for rural electrification in the developing
57 world. The town of Erijiyan, Nigeria is supplied with an average of 83 kW from a 3-bladed screw at the
58 nearby Arinta waterfall [14]. Okamura et al. (2015) proposed a pico-scale, off-grid ASG system to power
59 48 homes of a village in southern Tanzania. They proposed that the system be designed, manufactured,
60 and installed by locally-based companies to reduce costs and improve familiarity with the technology
61 [15]. The literature contains a French-language publication discussing modelling, development, and
62 testing of ASGs for use in typical waterfalls in the Democratic Republic of Congo [16].

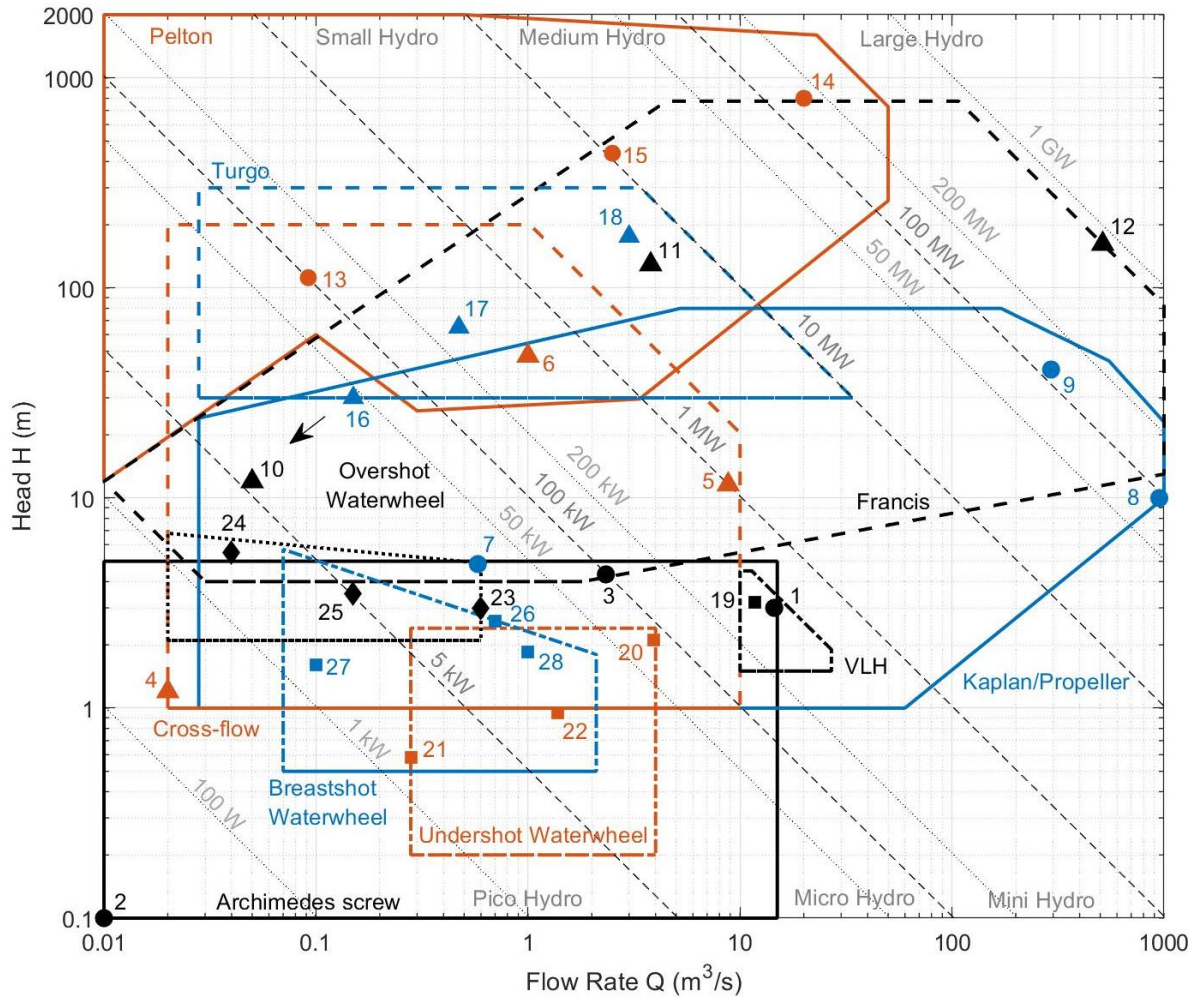
63 It was suggested that ASGs are good candidates for distributed microhydro scenarios. In some
64 developing countries there is a recognition of the need for more decentralized power generation [17].
65 There are also successful examples of hydropower community groups that may serve as a model for
66 future developments; the hydropower scheme at Ruswarp (UK) is successfully operated by the local
67 Whitby Esk Energy community group.

68 The precedents set by existing installations in the developed and developing world can help to
69 optimize the performance of Archimedes screw generator powerplants – improving efficiency and

70 reducing cost by experience. This article explores three successful ASG hydropower schemes to evaluate
71 their performance, documenting data that is crucial for future model developments.

72 **1.1. Archimedes screw generators**

73 Archimedes screw plants are small-scale hydropower plants that usually operate as diversion (or
74 “run-of-river”) schemes. Traditionally, ASGs operate at sites with low head ($H < 3.5\text{ m}$) and moderate
75 flow rates ($Q = 0.25\text{ to }5\text{ m}^3/\text{s}$) [3][18] (Fig. 2). However, more recent plants have installed and
76 operated much larger screws successfully, suggesting that this operating envelope could be extended to
77 include heads of about $H = 5\text{ m}$, and flow rates up to about $Q = 15\text{ m}^3/\text{s}$ [19]. For comparison, the
78 operating envelope of ASGs is plotted with other common hydro-electric technologies in Fig. 2,
79 including: Cross-flow, Kaplan, Francis, Pelton, Turgo, and Very Low Head (VLH) turbines, as well as
80 undershot, overshot, and breast-shot waterwheels. Some real-world example powerplants are
81 highlighted on the plot for reference, with details provided in Table 1.



82
 83 **Figure 2: Operational ranges of various hydro-generating technologies [18]. Solid lines indicate the operating envelope, the**
 84 **coloured dots are examples of the operating points of example hydropower plants. Data from various sources [20,21,30–**
 85 **39,22,40–44,23–29].**

86 Hydro turbines are often distinguished as either reaction or impulse turbines. Reaction turbines
 87 produce torque due to the pressure variations in a moving fluid, as such they tend to be used for larger-
 88 scale systems with higher flow rates and comparatively lower head. Francis turbines and propeller-type
 89 turbines are examples of reaction turbines. The Kaplan turbine is the most common variant of propeller-
 90 type turbines, and the very low head (VLH) turbine is a variant of the Kaplan turbine that is used for low
 91 head hydro sites. Impulse turbines generate torque by converting the impulse of a moving fluid into
 92 mechanical motion. They utilize comparatively high heads and correspondingly high static pressure
 93 differences to produce a high velocity fluid jet in the turbine; the high velocity fluid imparts its

94 momentum on the turbine rotor. Examples of common impulse turbines include Pelton, Turgo, and
 95 Cross-flow turbines.

96 Neither waterwheels nor ASGs are hydro turbines since they are not enclosed in pressure casings
 97 and do not use the same momentum mechanisms to convert fluid pressure into mechanical energy.
 98 Rather, both the screws and waterwheels compartmentalize water into discrete volume (“buckets”) and
 99 convert the static pressure of the bulk volume into mechanical power.

100 Table 1 provides more details of the example power plants defined in Fig. 2. Generally, three sites
 101 were shown for each device on the figure to provide the context of a comparatively large, moderate,
 102 and small instance of each device.

103 **Table 1: Index of example hydropower plants from Fig. 2. Turbine types are identified as follows: Archimedes screw**
 104 **generators (A), Crossflow turbines (C), Kaplan/Propeller type turbines (K), Francis turbines (F), Pelton turbines (P), Turgo**
 105 **turbines (T), VLH turbines (V), Undershot waterwheels (U), Overshot waterwheels (O), and Breast-shot waterwheels (B).**
 106 **Data from various sources [20,21,30–39,22,40–44,23–29].**

Site Number	Turbine Type	Site Identification	City	Country	Q (m ³ /s)	H (m)	Reference
1	A	Widdington	Linton-on-Ouse	UK	14.5	3	https://www.landustrie.nl/
2	A	PicoPica	Ena	Japan	0.01	0.1	http://www.unido.or.jp/en/technology_db/5276/
3	A	Buckfast Abbey	Buckfastleigh	UK	2.33	4.33	https://www.renewablesfirst.co.uk/
4	C	Universiti Kebangsaan Malaysia	Bangi	Malaysia	0.02	1.2	Razak et al. (2010)
5	C	Elora	Elora	Canada	8.8	11.6	http://www.ossbergerhydro.com/projects.html
6	C	Aspen	Aspen	Colorado	1	47.5	http://www.ossbergerhydro.com/projects.html
7	K	Novarina	-	-	0.58	4.85	Quaranta (2020)
8	K	Engineer Sérgio Motta Dam	São Paulo	Brazil	950	10	https://www.ge.com/renewableenergy/
9	K	Peixe Angical	Peixe	Brazil	293.3	40.9	http://voith.com/
10	F	HS Dynamic	Hong Kong	China	0.05	12	https://www.hs-dynamics.com/
11	F	Candelaria	Chisec	Guatemala	3.79	129.45	https://www.gilkes.com/
12	F	Jhimuk Hydropower Plant	Khaira	Nepal	513.9	162	https://nwrmap.info/hydropower/
13	P	Blair Castle	Blair Atholl	UK	0.092	112	https://www.gilkes.com/
14	P	La Esmeralda Dam	Chivor	Columbia	20	800	https://www.hydropower.org/case-studies/
15	P	Hidroeléctrica Choloma S.A	Guatemala City	Guatemala	2.5	438	https://www.gilkes.com/
16	T	Balbeg	Inverness	UK	0.15	30	http://www.highlandeco.com/
17	T	Grytviken	Grytviken	South Georgia	0.472	65	https://www.gilkes.com/
18	T	Pungwe B	Honde Valley	Zimbabwe	3	176	https://www.gilkes.com/
19	V	Isola Dovarese HPP	Isola Dovarese	Italy	11.75	3.2	http://www.vlh-turbine.com/
20	U	Queen's Mill	Castelford	UK	3.9	2.1	Quaranta (2020)
21	U	Undershot Waterwheel	-	-	0.28	0.58	Quaranta (2020)
22	U	Treviso Waterwheel	Treviso	Italy	1.38	0.95	Quaranta (2020)
23	O	Hydrowatt	-	-	0.6	3	Quaranta and Revelli (2018)
24	O	Cooperation project MAE-FAO	-	Italy	0.04	5.5	Quaranta and Revelli (2018)
25	O	Alan Stoyel and Graham Hackney	-	-	0.15	3.5	Quaranta (2020)
26	B	Moter Saenger	-	-	0.7	2.6	Quaranta (2020)
27	B	Rigamonti Giovanni	Pusiano	Italy	0.1	1.6	Quaranta (2020)
28	B	The Mill of Borgo Cornalese	Villastellone	Italy	1	1.85	Quaranta and Revelli (2018)

107

108 The Widdington site in Linton-on-Ouse, York, United Kingdom hosts the largest Archimedes screw
109 generator in the world (at time of construction in 2017) [45]. Due to its large diameter ($D_o = 5\text{ m}$) it can
110 operate at flow rates of up to nearly $Q = 13.5\text{ m}^3/\text{s}$ to produce a nominal power of about 280 kW. The
111 smallest-scale Archimedes screw known to be installed for energy generation is the *PicoPica10* device
112 manufactured by Sumino Co., Ltd. (Ena City, Japan). It is a modular screw generator that can produce up
113 to 10 W of electrical energy; the same scale as some laboratory experiments in the ASG literature
114 [46,47]. The company also produces a 500 W variant. Consequently, the Archimedes screw may be
115 implemented as a pico-hydro, micro-hydro, or mini-hydro device effectively. The operating range of
116 Archimedes screws make the device suitable for off-grid, rural installations. The simple, robust design of
117 ASGs also make them candidates for on-site or localised manufacturing; Archimedes screws have been
118 manufactured since antiquity as pumps and the design has not significantly changed. However, the
119 optimization of ASG design for this wide range of operating conditions requires an accurate
120 performance-predicting model.

121 There are a few performance models in the literature that predict power production and various
122 power losses associated with ASG operation [2,48–52]. However, the models are mostly developed with
123 data from laboratory-scale installations and are not robustly tested against data from the entire
124 operating range of ASGs. Laboratory-scale experiments and data are documented in the literature
125 [46,47,58–67,49,68,50,51,53–57]. The literature also includes some assessments of full-scale ASGs
126 [2,15,69–72], but most lack sufficient instrumentation or documentation to make using their findings
127 feasible for model evaluation.

128 To resolve the lack of full-scale ASG performance data, the authors and their research partner were
129 planning to fully instrument new, planned installations in Canada. However, due to recent changes in
130 the political landscape, installing these Archimedes screw plants in a timely manner was no longer an

131 option. A survey and measurement campaign was then planned to gather some data from various
132 installations in the United Kingdom, where more than 60 ASGs are currently operating [13].

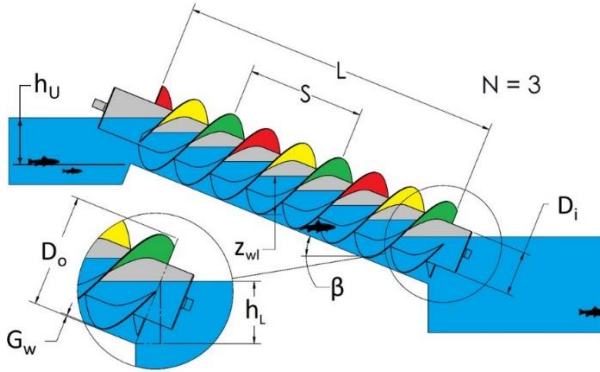
133 There are many more ASGs in several other European countries. The common (English) language
134 shared between the researchers and ASG operators was an important factor in the UK focus of this
135 study. This language commonality simplified establishing connections with plant operators to arrange
136 plant access, and in practice allowed much more detailed discussion of observations and technical issues
137 than would not have been possible without a shared language. Much of the significant work on
138 Archimedes screw generators and pumps has been reported in non-English literature, particularly in
139 German and Dutch. The authors do not have fluency in these languages, and have experienced first-
140 hand the difficulty of determining detailed technical information from literature in other languages,
141 even when those languages are relatively similar (i.e., English, German, and Dutch share similar
142 structure and vocabulary). The authors are aware of some Archimedes screw work in Japan, and have
143 reason to believe ASGs have also been studied in China but the significant language differences in these
144 cases have prevented truly thorough searches for this literature.

145 **2. Methods**

146 The research team conducted two trips to the United Kingdom to gather full-scale ASG
147 performance data. The first trip involved site visits to the powerplants shown in Table 2, while the
148 second trip focussed on three powerplants (Table 3). These two trips will be outlined in this section.

149 **2.1. Preliminary Surveying Trip**





150 The first trip occurred in April 2019 to survey and measure datum points throughout a series of
151 power plants, measure screw dimensions as per Fig. 3, and collect power production values as well as
152 operating ranges of flow, head, and rotation speed.



153
154 **Figure 3: Dimensions of an Archimedes screw [19].**





155 Most of the screw powerplants in the United Kingdom experience highest flow and head in the
 156 winter months. The preliminary trip was carried out to identify optimal sites for a more detailed
 157 measurement campaign. Site-specific details were documented to help with further detailed
 158 measurements and to ensure the right equipment was available. Table 2 shows the sites visited, their
 159 locations, and various geometric and design parameters for reference.

160 Table 2: sites that were visited and surveyed on the first trip of the measurement campaign (1 of 3).

Site	Linton		Widdington		Goldsborough		Howsham Mill (1)	
Town	Linton-on-Ouse		Linton-on-Ouse		Knaresborough		Howsham	
Region	York		York		York		York	
Year Installed	2012		2017		2014		2007	
Approximate Coordinates	54°02'01.4"N 1°14'17.8"W		54°02'01.4"N 1°14'17.8"W		53°59'51.6"N 1°26'28.5"W		54°03'21.4"N 0°53'09.7"W	
Screw Dimensions	D _o	3 m	5 m	-	-	-	-	-
	D _i	-	2.22	-	-	-	-	-
	L	8.5	7.5	-	-	-	-	-
	S	-	5.0625	-	-	-	-	-
	N	4	4	4	4	4	4	4
	β	22°	22°	22°	22°	22°	22°	22°
Rated/Expected Levels	Q	4.5 m ³ /s	14.5 m ³ /s	-	-	-	-	-
	ΔH	3.2 m	3 m	2 m	2 m	2 m	2 m	2 m
	P	101 kW	355 kW	100 kW	100 kW	100 kW	24 kW	24 kW
Photograph								





161

162 Table 2 (continued): sites that were visited and surveyed on the first trip of the measurement campaign (2 of 3).

Site	Howsham Mill (2)		New Mills / Torrs Hydro		Church Minshull		River Dart Country Park	
Town	Howsham		New Mills		Church Minshull		Ashburton	
Region	York		Derbyshire		Cheshire		Devon	
Year Installed	2017		2008		2018		2008	
Approximate Coordinates	54°03'21.4"N 0°53'09.7"W		53°22'01.2"N 2°00'25.2"W		53°08'37.6"N 2°29'51.7"W		50°31'02.7"N 3°47'20.6"W	
Screw Dimensions	D _o	-	2.6	-	-	-	2.24	-
	D _i	-	1.22	-	-	-	1.12	-
	L	-	9	-	-	-	11.54	-
	S	-	-	-	-	-	2.308	-
	N	-	-	-	-	-	3	-
	β	-	22°	22°	Variable	Variable	22°	-
Rated/Expected Levels	Q	-	-	-	-	-	-	-
	ΔH	-	3 m	3 m	-	4 m	4 m	4 m
	P	55 kW	63 kW	63 kW	80 kW	80 kW	65 kW	65 kW
Photograph								

163

164 **Table 2 (continued): sites that were visited and surveyed on the first trip of the measurement campaign (3 of 3).**

Site	Buckfast Abbey		Totnes		Tipton		Windsor / Romney Weir		
Town	Buckfastleigh		Totnes		Lancercombe		Windsor		
Region	Devon		Devon		East Devon		Berkshire		
Year Installed	2012		2015		2010		2013		
Approximate Coordinates	50°29'33.2"N 3°46'31.3"W		50°26'19.2"N 3°41'24.2"W		50°43'29.4"N 3°17'16.8"W		51°29'17.2"N 0°36'22.0"W		
Screw Dimensions	D _o	2.5	m	3.7	m	2	m	4	m
	D _i	1.22	m	1.62	m	0.914	m	1.6	m
	L	10.562	m	9.1	m	7.261	m	5.85	m
	S	2.5	m	3.71	m	2.34	m	4	m
	N	4	-	4	-	3	-	5	-
	β	26	°	22	°	22	°	6 to 20	°
Rated/Expected Levels	Q	-	m ³ /s	2 x 6.5	m ³ /s	1.5	m ³ /s	2 x 10.4	m ³ /s
	ΔH	-	m	3.45	m	2.6	m	2	m
	P	84	kW	2 x 160	kW	28	kW	2 x 157	kW
Photograph									

165

166 Relative heights of water surfaces and plant components were measured using conventional

167 surveying techniques. Two individuals were necessary for these measurements: one to operate a

168 manual transit, and the other to place and hold-steady a survey pole at desired locations throughout the

169 plant. Desired locations typically included a point on the inlet and outlet of the screw, a point of the

170 screw frame, and the location of gates and other fixed items at the installation (i.e. sluice gate,

171 generator, grid-connection, etc.). Once the site survey was carried out, the water levels at the plant

172 were measured to compare and recalibrate on-site sensors. The site was then well documented with

173 written notes, photography, and videography.

174 Notes were taken about screw performance, maintenance, operating issues, and various other site-

175 specific details. Photography was used to document the locations of each measurement and to

176 document various design aspects of the plants. Video footage was used to measure the rotation speed

177 of the screw, document screw filling and to capture the wave-action exhibited in the screw buckets as

178 they translate along the length of the screw.

179 A Sontek FlowTracker2 handheld Acoustic Doppler Velocimeter (ADV®) (Sontek, San Diego, USA)
180 was made available during the surveying trip in case water velocity could be measured freely, but it was
181 found that measuring velocity and flow rate would require further access from site owners and
182 operators. (Access was granted for the next trip during the high flow season.)

183 Flow rate of water in channels is difficult to measure with low uncertainty, and in these studies the
184 on-site flow rate measurements were the most difficult to gather accurately. Most sensors, like the
185 Sontek ADV® sensor, measure velocity at a point. Both the longitudinal bedform and cross-sectional area
186 of the inlet channel affect overall channel flow rate. A long, straight channel will exhibit predictable
187 near-uniform flow rates for given water levels. However, on the other extreme, a short channel with a
188 sharp bend will be more visibly subjected to minor losses and flow regime changes.




189 A range of velocity measurements were required at points across a cross-sectional mesh to
190 determine the overall volume flow rate of water through the screw. If these measurements were taken
191 in a long, straight channel then a rating curve could be developed to compare water depth (or stage) to
192 the flow rate (or discharge) in the channel. Afterwards, the measurements were spatially integrated to
193 determine the flow rate. There was a trade-off between grid resolution and accuracy during the flow
194 measurements: a finer mesh yielded more accurate results but required significantly more time to
195 measure. Measuring time played a roll in the error, however, since the flow rate commonly changed as
196 time passed; biasing the flow measurement.

197 Altogether, accurate flow measurements were required to be taken as quickly as possible along the
198 cross-section of a long, straight inlet channel for the results to be acceptable to any degree of accuracy.
199 During the surveying trip, sites were identified that would make viable candidates for flow rate
200 measurements. These sites included: River Dart, Buckfast Abby, and Tipton.

201 **2.2. Flow Rate and Performance Measurement Trip**

202 Due to availability and further contact with another site owner, the flow rate and performance
 203 measurement trip was planned for February 2020 with visits to Buckfast Abbey and an installation
 204 owned and operated by a community hydro group, Whitby Esk Energy, in Ruswarp, North Yorkshire. A
 205 follow-up visit was also planned for the installation in Totnes due to the regular and significant variation
 206 in outlet water level caused by tidal variation at the screw outlet. Table 3 indicates the sites visited, their
 207 locations, and their geometric and design parameters.

208 **Table 3: sites that were visited during the flow and performance measurement trip.**

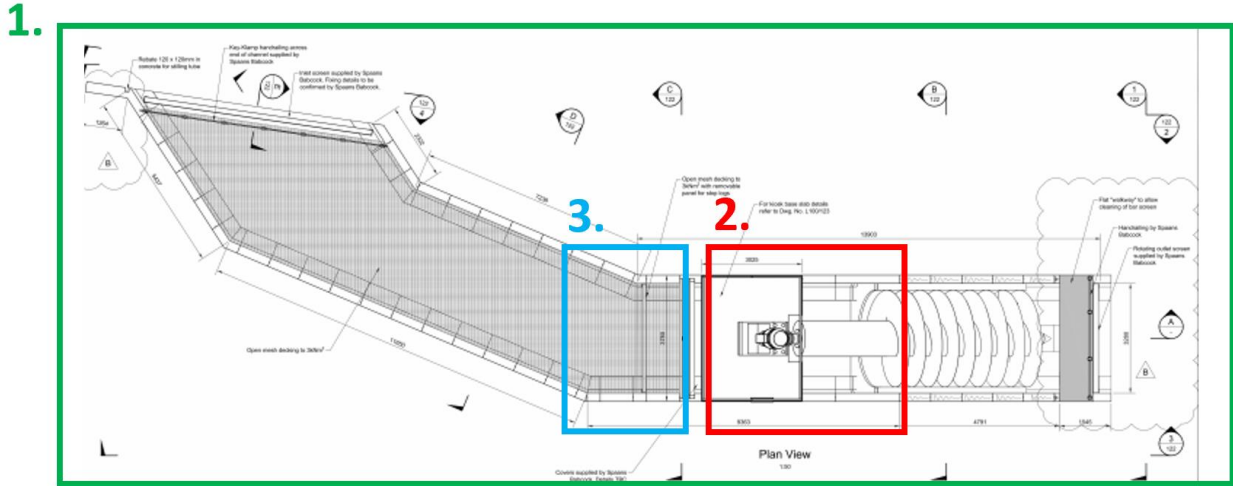
Site		Ruswarp		Buckfast Abbey		Totnes	
Town	Region	Ruswarp North Yorkshire		Buckfastleigh Devon		Totnes Devon	
Approximate Coordinates		54°28'05.5"N 0°38'01.1"W		50°29'33.2"N 3°46'31.3"W		50°26'19.2"N 3°41'24.2"W	
Screw Dimensions	D _o	2.9	m	2.5	m	3.7	m
	D _i	1.2	m	1.22	m	1.62	m
	L	5.1167	m	10.562	m	9.1	m
	S	3.07	m	2.5	m	3.71	m
	N	3	-	4	-	4	-
	β	22	°	26	°	22	°
Exhibited Levels	Q	3.95	m ³ /s	2.33	m ³ /s	-	m ³ /s
	ΔH	0.963	m	4.33	m	-	m
	P	27.8	kW	74.3	kW	329.2 (total)	kW
Photograph							

209
 210 The measurement goals and methods for each of these site visits will be discussed in further detail
 211 within this section.

212 **2.2.1. Ruswarp Hydro**

213 Preparations and planning for the Ruswarp Hydro site visit were carried out via email
 214 communication with the community hydro group, Esk Energy (Yorkshire). Since this site had yet been
 215 visited in-person, the same surveying equipment from the first trip was brought and used to confirm
 216 elevations reported in construction drawings provided by Esk Energy. Fig. 4 illustrates the order and

217 locations of the measurements taken at Ruswarp overlaid on an overhead view powerplant drawing.
218 Drawings of the Ruswarp installation are shown in the appendix.



219
220 **Figure 4: Ruswarp Hydro measurement campaign started with a site survey (1), then measurements within the forebay (2),**
221 **and finally flow measurements in the inlet channel upstream of the forebay's sluice gate (3).**

222 After a tour of the site, the manual transit was placed between the inlet channel and the fish pass
223 to measure heights of points along the inlet channel, screw frame, and the outlet (Fig. 5). Water levels
224 were measured, and photos were taken at each measurement location for reference. A video was taken
225 of the screw during operation to verify rotation speed and conditions within the bucket. An acoustic
226 range finder sensor was installed on the screw to gather a measurement of the bucket fill height during
227 operation; however, the sensor failed during operation, so these measurements were not recorded.



228
229 **Figure 5: view of the transit and survey pole. One individual operated the transit while the other steady the survey pole to**
230 **record a measurement.**

231 The water levels and data points measured with the transit and survey pole were used to verify
232 water levels, re-calibrate sensors, and as a reference point so that site-recorded data could be used to
233 gather further measurements.

234 Next, the forebay sluice gate was closed, and the screw was drained so that measurements could
235 be taken of the screw and the plant for reference (Fig. 6). These measurements were used to verify
236 screw dimensions and to add a reference point for upper water level (forebay water level)
237 measurements. Future measurements may be made with the monitor camera shown in Fig. 6b. It is
238 interesting to note that the screw was originally designed at a higher point along the weir, however after
239 a design review the screw was shortened. The trough was not shortened until after commissioning when
240 it was found that it contributed to high levels of turbulence at the screw's inlet. The shortening of these
241 components is evidenced by the seam on the inner cylinder and the longer baffle on the upper right side
242 of the trough in Fig. 6a. During operation, the screw exhibited larger levels of overflow due to the
243 shortening and lower position along the weir.



244 **Figure 6: measurements were taken inside the screw's forebay for reference with upper water level measurements. Note in**
245 **the left image (a) that the baffle on the upper right side of the trough extends past the screw. The right image (b)**
246 **demonstrates monitor camera placement for tracking inlet water level and bucket filling.**

248 Placement of the top end of the screw on the weir is critical to screw performance. If the screw is
249 placed too low, upper water levels will be too high and cause flooding to the first buckets of the screw.
250 In the event this happens, operators are forced to lower the sluice gate at the inlet to let less water in –

251 reducing overall plant efficiency. If the screw inlet is placed too high along the weir the initial bucket will
252 be starved during its filling operation and the plant will perform with lower efficiencies due to under-
253 filling.

254 Finally, the flow measurements were carried out at the site. Point source velocity was measured
255 with the Sontek ADV[®] sensor along a grid at the cross-section of inlet channel just upstream of the
256 forebay's sluice gate (shown in Fig. 7). A tape measure was placed along the channel cross-section at this
257 location so that the x-position of the measurements could be maintained and documented (Fig. 7b). The
258 Sontek ADV[®] sensor was equipped with an adjustable pole that was used to measure and control the y-
259 position of the sensor (Fig. 7c). Velocity measurements were taken at 13 horizontal and 7 vertical
260 positions across the inlet cross section (cf. Fig. 13) to characterize the velocity profile. With the location
261 of the velocity measurement on the grid known, the velocity was recorded and integrated spatially to
262 determine the flow rate of the inlet channel. Plant settings were adjusted so multiple flow rates could
263 be measured.

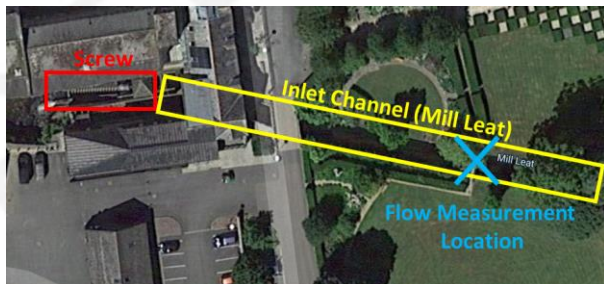


264
265 **Figure 7: view of sluice gate and flow measurement area from the upstream opening of the inlet channel (left, a), measuring**
266 **tape placed along the inlet channel's cross-section to measure velocity and known horizontal locations (middle, b), and two**
267 **individuals with the Sontek ADV[®] sensor and extended pole ready to measure flow velocity through an open inlet channel**
268 **grate (right, c).**

269 Due to the bend in the channel, one side of the channel had a very high current, while the other
270 side of the channel had a near stall point. It was very difficult to hold the Sontek ADV[®] sensor in place
271 due to the rushing water, so the sensor pole was redesigned for the measurements at Buckfast Abbey.

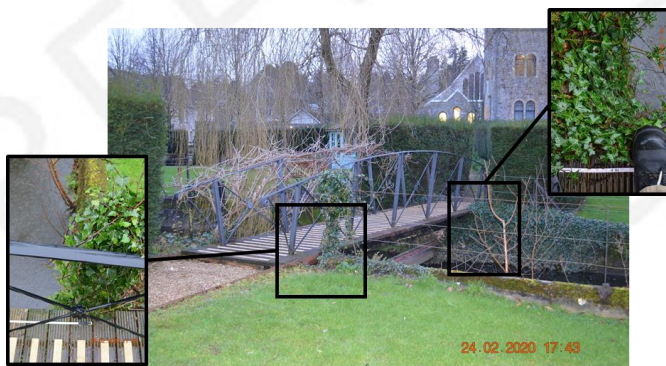
272 2.2.2. Buckfast Abbey

273 Similar methods were applied to measure flow rate at Buckfast Abbey. Upon arrival at the site, the
274 inlet channel (or mill leat) was examined to select the best location to measure a velocity grid. A birds-
275 eye satellite image of the site is shown in Fig. 8 with the screw installation, inlet channel, and flow
276 measurement locations highlighted. Drawings of the Buckfast Abbey installation are shown in the
277 appendix.



278
279 **Figure 8: satellite view of Buckfast Abbey site showing screw (red), inlet channel (yellow), and flow measurement (blue)**
280 **locations.**

281 Upon arrival at the site, the sluice gate at the top end of the inlet channel was set to an 80%
282 opening, allowing the full rated flow to pass through the inlet channel. Velocity was measured across a
283 grid along the cross-section of the channel identified in Fig. 8. It was measured at 15 equally spaced
284 horizontal locations and 4 heights along the cross section spanning from 65 to 460 cm and 0 to 145 cm,
285 respectively. It is noted that the channel cross-section was a small-angled trapezoid and that there was
286 significant debris build-up along the channel bed and walls – which noticeably impacted near-wall
287 velocities (Fig. 9).



288

289 **Figure 9: debris on channel side walls affected near-wall velocities.**

290 After the full grid was measured for an 80% sluice gate opening, the gate was set to a 70% opening.
291 The site-recorded water level measurements were plotted in real-time to evaluate when the plant
292 began to operate at equilibrium under the new inlet conditions – it usually took 35 minutes for the plant
293 to “settle” under the new flow rate. Once equilibrium was met, 3 points were measured in the free-
294 stream region of the flow measurement location’s cross-section. Since the inlet channel was sufficiently
295 long and straight, the three freestream measurements were used to scale the other measurements of
296 the grid from the initial velocity measurements along the grid (i.e. the 80% sluice gate opening
297 measurements), saving measurement time while still capturing a relatively accurate flow rate reading.
298 This was repeated for a total of 6 different flow rates corresponding to 6 different sluice gate opening
299 values.

300 It was noted that the outlet water level varied significantly throughout the tests. At the start of the
301 test, the water level was near 80% submersion of the bottom end of the screw (Fig. 10a). Upon
302 completion of the flow measurements the outlet water level was much higher than 120% outlet
303 submersion (Fig. 10b).



304
305 **Figure 10: outlet water level near the start of measuring (left, a) and at the end of testing (right, b).**

306 This variation in outlet water level had no impact on the flow measurements, but it did lead to a
307 decrease in plant performance since the overall head drop decreased. Practically, measurements of
308 varying outlet water level are very difficult to measure at full-scale installations since water levels are
309 usually steady throughout a single day. While southern England does house very “flashy” watersheds, it

310 was quite lucky that measurements were able to be gathered with large outlet water level variation
311 throughout a single day since it will help to more robustly evaluate model performance. Only plants with
312 tidal variations at their outlet commonly experience drastic outlet water level variation throughout a
313 day – this is the reason that a follow-up visit was scheduled at the Totnes power plant.

314 2.2.3. Totnes

315 The power plant in Totnes houses twin parallel screws that experience tidal variations in their
316 outlet water level. The power plant is shown in a satellite view in Fig. 11. Drawings of the Totnes
317 installation are shown in the appendix.



318 **Figure 11: Totnes power plant has a short inlet and significant outlet water level variation due to impacts of ocean tides.**
319 **North is upward. The plant is at lower right on the south end of the weir. Flow is from left to right.**
320

321 During the site visit at Totnes, the inlet channel had large debris build-up which precluded a survey
322 using the flow meter to measure the volume flow rate at the inlet. The inlet of this plant is quite large,
323 and it would have been difficult to conduct an inlet survey even in optimal conditions without debris
324 present. Instead, data was gathered from the on-site computer terminal that recorded power readings
325 for both screws, and the up- and downstream water levels. Data was collected manually for the a few
326 hours of the tidal cycle – enough to see it level-off (Fig. 12). This data was stored for post processing
327 later.



328
329

Figure 12: outlet water level near site arrival time (left, a) and at the end of the tidal cycle (right, b).

330 Visually, the outlet of the screw was about 90% submerged upon arrival, and only about 60%
331 submerged at the end of the tidal cycle. The bankside screw (the nearest screw in Fig. 12) had a recent
332 flying failure and was operating with $N = 3$ blades rather than $N = 4$. The riverside screw (the
333 further screw in Fig. 12) was operating with all four blades, as designed. The authors noted that the
334 power production of the two screws was nearly identical, with the bankside screw only producing an
335 average of about 3.2% less power. This indicates that the simple, robust design of the screw can operate
336 effectively when experiencing severe failure.

337 3. Results and Discussion

338 First, some general observations will be presented from both the surveying trip and the flow and
339 performance measuring trips. Afterwards, results will be presented in a similar order to the methods
340 section starting with Ruswarp Hydro, then Buckfast Abbey and Totnes. Finally, Section 3.5. shows some
341 preliminary results for a follow-up scaling study that will use a computational fluid dynamics (CFD)
342 model evaluated for accuracy against this full-scale data.

343 3.1. General Archimedes screw plant observations

344 Eleven Archimedes screw sites were visited in total in the United Kingdom during two trips. It was
345 noted that most of the sites had inclination angles of $\beta = 22^\circ$, with the exception being Buckfast Abbey,
346 Church Minshull, and the Romney Weir. Buckfast Abbey was an exceptionally long screw since it

347 operated at a relatively high head (nominal $H = 4.2\text{ m}$) location that had historically powered a
348 waterwheel. As such, it was installed at a steeper inclination angle ($\beta = 26^\circ$) to allow for a shorter
349 length to reduce capital costs. The installations at Church Minshull and the Romney Weir had variable
350 inclination angles. The plant operator had found highest efficiencies occurred at $\beta = 22^\circ$ for the
351 Romney Weir installation [73].

352 Most Archimedes screws were designed with $N = 4$ blades, the exceptions being the River Dart
353 Country Park ($N = 3$), Ruswarp Hydro ($N = 3$), and the Romney Weir ($N = 5$). However, after follow-up
354 discussions with the builder, it was suggested that the Romney Weir would likely have produced more
355 power if it were designed with $N = 4$ flights [73]. It is also relevant to note that the $N = 3$ bladed plant
356 at the River Dart Country Park was the first screw plant installed in the UK and likely served as a proof-
357 of-concept.

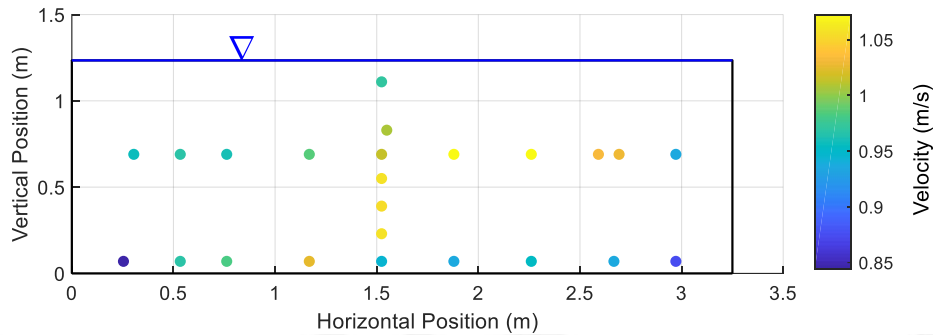
358 Aside from the installations at Tipton and Buckfast Abbey, all the screws visited operated with
359 variable speed. Tipton operated with a single speed induction generator to have a lower cost of
360 operation and more efficient mechanical-electrical energy conversion.

361 Many sites had low average flow rates but very high flood levels. The UK has very “flashy” water
362 systems and are prone to large variation in flow and water levels. It was common to find that even
363 variable-speed screw plants did not operate for long during low flow periods. Without variable-speed
364 controls, many plants would not run most of the time. Two plants (Linton and Church Minshull) had
365 water-proofed generators, and six plants had water-proofed generator rooms or enclosures
366 (Widdington, Goldsborough, Howsham Mill, Totnes, and Romney Weir). Though it was water-proofed,
367 the Totnes installation had fewer flooding concerns; as did the installations at The River Dart and
368 Buckfast Abbey, due to site topography. Ruswarp Hydro installed all flood-sensitive equipment 0.5 m
369 above the 100-year flood level provided by the UK’s Environment Agency.

370 **3.2. Ruswarp Hydro**

371 The velocity cross-sectional grid was spatially integrated to find the flow rate for each of the three
 372 settings carried out at Ruswarp Hydro. The measurement locations and observed velocities are shown in

373 Fig. 13.



374 **Figure 13: velocity profile along the channel’s rectangular cross-section for the first test. The free surface is shown along the**
 375 **top with a blue line. The horizontal channel bed and vertical side walls are shown as black lines. The horizontal position is**
 376 **measured from the south bank of the inlet channel and the vertical position is measured from the channel bed.**
 377

378 The power plant’s water levels (upper h_U , lower h_L , and overall head ΔH), measured flow rate (Q),
 379 generating torque (T), rotation speed (Ω), mechanical power produced by the screw (P_s), and the
 380 electrical power produced by the plant (P_e) are shown in Table 4.

381 **Table 4: Results of the measurements taken at the Ruswarp Hydro ASG plant on 22 February 2020.**

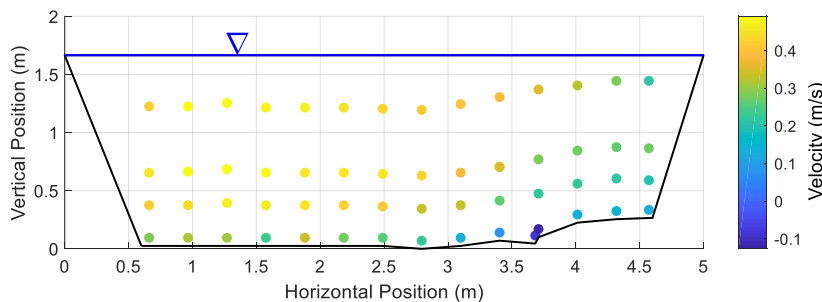
Test Number	ΔH (m)	h_U (m)	h_L (m)	Q (m^3/s)	T (Nm)	Ω (RPM)	P_s (kW)	P_e (kW)
1	0.963	1.234	2.030	3.949	11620	26.7	32.5	27.8
2	0.771	1.044	2.054	2.850	12693	20.0	26.6	22.3
3	0.768	0.826	1.878	1.770	14544	18.4	28.0	23.8

383 Based on the resolution of the velocity grid and the care taken during the measurements, the
 384 authors have confidence in the flow measurement of test number 1. However, due to time constraints it
 385 was not possible to measure tests 2 and 3 with the same resolution, and the authors are not certain of
 386 their validity. Due to the site’s complex channel geometry, a higher resolution sample grid was required
 387 to make accurate flow measurements. So, the authors caution that data points 2 and 3 have a high
 388 degree of uncertainty and might not be as appropriate for model evaluation.

389 With the torque and rotation speed known, the mechanical power at the shaft was calculated. The
390 mechanical power value (P_s) represent the mechanical conversion capabilities of the Archimedes screw
391 (i.e. its ability to convert hydraulic power into mechanical power for electrical generation). With the
392 water levels, power, and flow rates known, this full-scale dataset may be used to evaluate existing
393 models from the literature, and aide in future model improvements.

394 3.3. Buckfast Abbey

395 The results of the measurements taken at Buckfast Abbey are presented in this section. Firstly, the
396 velocity profile at the cross-section of channel where the flow measurements were taken is shown in
397 Fig. 14.



398 **Figure 14: velocity profile along the channel cross-section for the 80% open sluice gate case. The free surface is shown along**
399 **the top with a blue line. The horizontal channel bed and vertical side walls are shown as black lines. The horizontal position**
400 **is measured from the south bank of the inlet channel and the vertical position is measured from the channel bed.**
401

402 As mentioned in Section 2.2.2. the debris on the side walls changed the velocity profile. The change
403 in vertical position is due to variation in the height of the channel bed caused by debris buildup. The
404 velocity profile indicates that the right side of the channel (i.e. 460 cm) had more debris than the left
405 side (i.e. 50 cm). It also indicates that the debris on the channel bed on the bottom right side has a
406 higher roughness height than the left side, since the velocity change was more drastic.

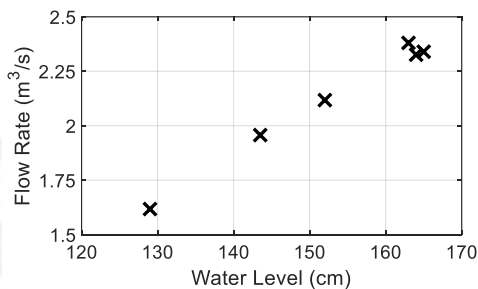
407 A total of six flow settings were measured during the campaign at Buckfast Abbey. As mentioned
408 previously, the first flow setting had its entire cross-sectional grid measured to develop the velocity
409 profile shown in Fig. 14. Subsequent flow settings were measured at three points along the free-stream

410 region of the grid, and the velocity profile of Fig. 14 was scaled based on these three points to
 411 approximate the curves for all other flow rates. The results of integration along the velocity profile
 412 shown in Fig. 14, and the integration of each scaled velocity profile are shown in Table 5.

413 **Table 5: Results of the measurements taken at the Buckfast Abbey ASG plant on 24 February 2020.**

Test Number	ΔH (m)	h_u (m)	h_L (m)	Q (m^3/s)	T (Nm)	Ω (RPM)	P_s (kW)	P_e (kW)
1	4.333	1.708	1.978	2.326	27564	28.8	83.0	74.3
2	4.076	1.735	2.261	2.340	25917	28.8	78.1	69.8
3	3.726	1.616	2.492	2.118	20800	28.8	62.7	55.7
4	3.374	1.391	2.620	1.618	12601	28.8	38.0	32.7
5	3.490	1.515	2.628	1.957	16950	28.8	51.1	44.9
6	3.725	1.713	2.591	2.380	22418	28.8	67.5	60.2

414
 415 The results of Table 5 are shown graphically in Fig. 15, relating the height of water at the point of
 416 measurement in the inlet channel to the flow rate in the inlet channel.

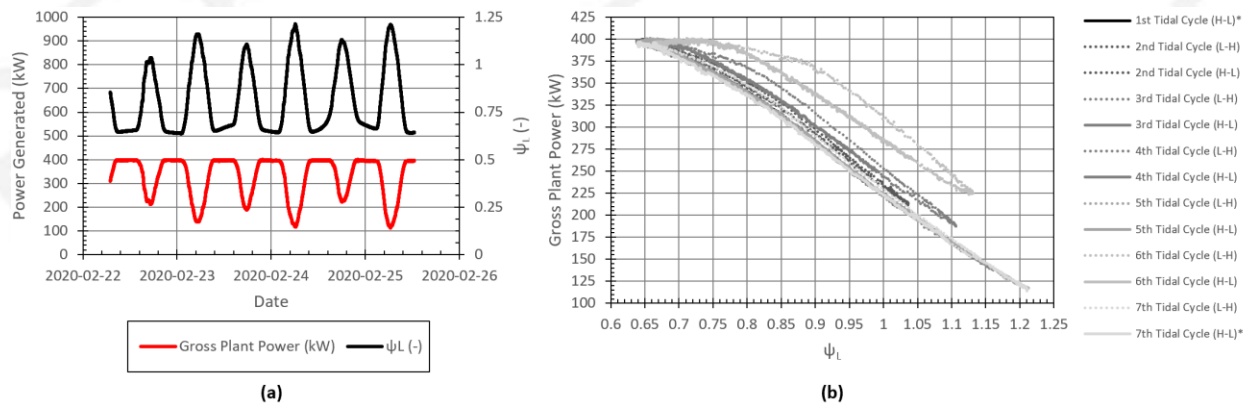


417
 418 **Figure 15: rating curve for Buckfast Abbey inlet channel.**

419 As expected, a predictable trend existed between the water level and the discharge in this long,
 420 straight inlet channel. This suggested that future flow rate measurements can be approximated with a
 421 measurement of the water level at this location in the channel. Like the Ruswarp measurements, the
 422 water levels, flow rates, speed, and power readings of this site were all measured and may thus be used
 423 for model development and evaluation.

424 **3.4. Totnes**

425 The final site examined during the second measurement campaign was the screw plant in Totnes.
 426 As mentioned, measuring the flow rate was not possible, but the authors did document some
 427 interesting data collected from other on-site instrumentation. Figure 16 shows the outlet submergence
 428 (ψ_L) and gross plant power production capabilities during six complete tidal cycles. Figure 16a shows
 429 ten-second increments of the two measurements, while Figure 16b directly compares the two.



430
 431 **Figure 16: Plot (a) demonstrates the tidal variation effect on plant power production (red) and dimensionless outlet**
 432 **submergence, ψ_L (black). The same data is directly compared in plot (b).**

433 The authors note that UK guidelines require plants to be designed with an upper limit of power
 434 production – in this case, Totnes did not exceed 400 kW of generated power, even if conditions would
 435 have allowed greater output. With that in mind, it is apparent that the power output exhibits a
 436 dependency on outlet submergence, although apparent hysteresis in Figure 7b indicates the power
 437 output is not solely influenced by outlet water level.

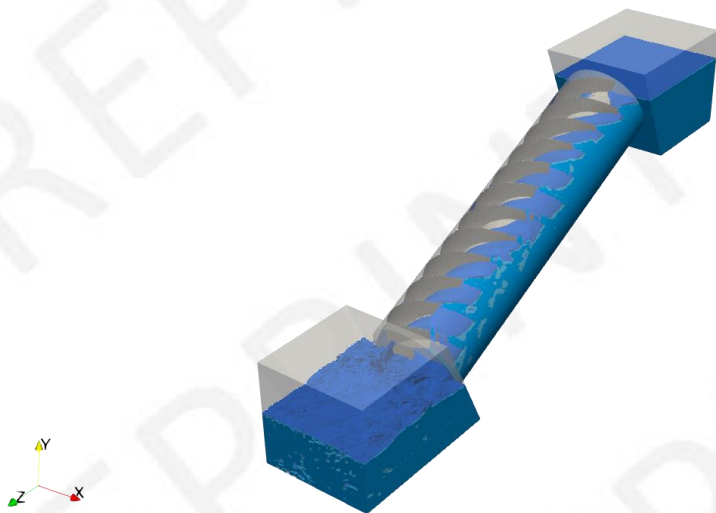
438 The amplitudes varied slightly between the two curves in Figure 17a: the second last “peak” in
 439 outlet submergence had a value of $\psi_L \approx 1.13$, and the following “peak” (i.e. the last peak) nearly
 440 reached $\psi_L = 1.25$. This is a change of about 110.6%, while the corresponding power production values
 441 changed from $P_e \approx 210 \text{ kW}$ to $P_e \approx 100 \text{ kW}$, meaning a change of about -210%.

442 Though not enough information was gathered to make this data useful for model development
443 and evaluation, it is still very valuable to directly observe the effect of outlet water level on performance
444 of a full-scale plant.

445 **3.5. Preliminary work evaluating a full-scale CFD model**

446 As in many other fields, there is notable lack of useful measured data in the literature for evaluating
447 the performance of numerical or parametric models of Archimedes screw plants. This lack of evaluation
448 data is one of the most significant limitations on further developing such models, and was the primary
449 motivation for this study.

450 As an example, the data gathered from the Ruswarp and Buckfast Abbey power plants was used to
451 evaluate a full-scale dynamic CFD model of an Archimedes screw generator plant shown in Fig. 17. The
452 model is discussed in greater detail in the literature [74].

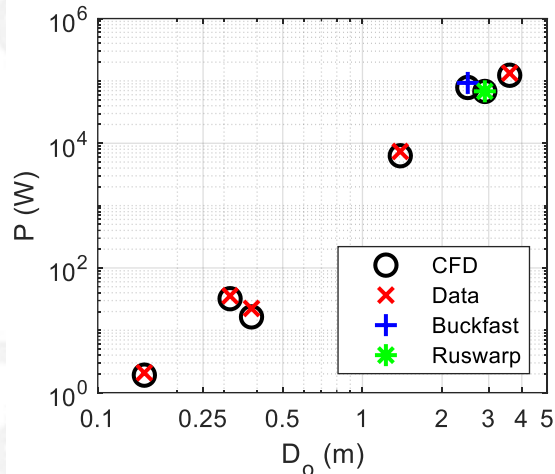


453

454 **Figure 17: full-scale dynamic CFD model of an ASG power plant.**

455 The flow rate and rotation speed measured on-site were used to set up the model, and the model
456 was run until it converged. The averaged values were then plotted against experimental observations

457 and measured values in Fig. 18 to evaluate the accuracy of the CFD simulations. The additional data
458 points in Fig. 18 come from laboratory studies [49,50,62,74] and other field studies [50,61,72].



459 **Figure 18: comparison of CFD simulation results to measurements and experimental data. Experimental data gathered from**
460 **Buckfast Abbey and Ruswarp Hydro are highlighted.**
461

462 The CFD model successfully predicted screw performance across a wide range of installation sizes.
463 A log-scale plot was needed to show the wide range of the data points; however, it minimized the
464 apparent differences between the simulated and measured values. Differences were minor, but still
465 present. Some of the minor discrepancies between the simulated and measured values may be due to
466 the differences in geometry between the simulated and measured screws. The CFD simulation models a
467 geometrically perfect screw while the real-world screw plants and laboratory screws have
468 imperfections. For example, the gap width of the simulation is perfectly uniform, where a real screw will
469 have some differences across its perimeter within manufacturing tolerances, causing a slightly variable
470 gap width. The CFD simulation domain was scaled across the range, and so differences in the geometry
471 of the delivery channels of the real-world screws were not modelled. For example, both Buckfast and
472 Ruswarp have a smoothed, inlet fairing treatment to reduce hydraulic losses. However, the CFD
473 simulation used the sharp inlet transition between the upper reservoir and trough shown in Fig. 17 and
474 did not match the real-world reservoir geometry.

475 **4. Design Insights**

476 The authors have compiled this section to offer some practical insight based on the site surveys,
477 measurements, and experimentation in this study. It offers some insights and considerations for ASG
478 powerplant design.

479 **4.1. Power production estimates**

480 There is no simple way to accurately approximate screw power; however, Nuernbergk [75] created
481 a table to estimate the screw's optimal mechanical efficiency based on its outer radius and rotation
482 speed. The table shows the mechanical efficiency of Archimedes screws (converting pressure to
483 mechanical torque); ASGs have high mechanical efficiencies, upwards of 80%. ASGs perform with
484 relatively high river-to-wire efficiencies as well; some plants perform with river-to-wire efficiencies
485 higher than 75%. Once the mechanical efficiency was approximated, the optimal shaft power could be
486 estimated by:

$$487 \quad P_s = \eta P_a = \eta \rho g H Q$$

488 Full performance prediction models are more complicated. Nuernbergk outlined various power
489 prediction and power loss models in the German-language book *Wasserkraftschnecken* [75], and Lubitz
490 [49] presented a performance model with subsequent publications to update power loss modelling
491 techniques [50,62,76]. Accuracy improvements are underway using the full-scale powerplant data
492 gathered in this study. Generally, the model used flow rate, upper and lower water levels, rotation
493 speed, and the screw's geometry to estimate power. Functionally, the model calculates the average fill
494 height in the screw's buckets, then integrates along the blades to convert the blade pressure to a torque
495 about the rotational axis. The torque is then multiplied by the average number of buckets in the screw
496 and the rotation speed to find the total power production; more details may be found in the cited
497 literature.

498 **4.2. Inclination angle**

499 It is most common to observe screw generators with inclination angles of $\beta = 22^\circ$, this is also a very
500 common inclination for screw pumps. The Romney Weir installation has a varying inclination angle; the
501 plant owner documented screw performance under varying inclination angles and found the optimal
502 inclination to be approximately $\beta = 22^\circ$. Numerical simulations suggest that the optimum inclination
503 angle is between $\beta = 20^\circ$ to 25° , supporting these claims.

504 However, the authors suggest that the optimum inclination angle is site-specific, and dependent on
505 other screw geometry variables (e.g. number of blades, etc.). Numerical simulations showed that the
506 simulated screw geometry with $N=3$ blades performed at its highest efficiency when $\beta \approx 15^\circ$. The same
507 study found the optimal efficiency of a 4- and 5-bladed variant was at $\beta \approx 20^\circ$ and 25° , respectively.

508 Powerplant design is usually based on return on investment (ROI). Some screws that have higher
509 head drops may be installed at steeper inclinations to lessen the cost of civil works. For example,
510 Buckfast abbey has a proportionally large head drop, which is likely why it was installed with an
511 inclination of $\beta = 26^\circ$, rather than the conventional 22° .

512 **4.3. Diameter ratio**

513 The diameter ratio of screw pumps was theorized to be optimal at $D_o/D_i = 0.54$ [77]. The optimum
514 was found by maximizing the volume within a screw's bucket, so it is applicable to screw generators as
515 well. Archimedes screw generators have commonly been installed with diameter ratios to approximately
516 match the theoretical maximum. Usually, the inner cylinder of the screw is a standard sized pipe, and
517 the blades are built to match the outer diameter requirements. So, in practice, the diameter ratio is a
518 target value.

519 **4.4. Pitch and screw length**

520 In practice, most screws have a pitch-diameter ratio of $S/D_o = 1$. Laboratory-scale experimentation
521 found agreement with this ratio [46]; three screws with $S/D_o = 1.4, 1.0,$ and 0.8 were compared, the
522 screw with $S/D_o = 1.0$ produced power most efficiently.

523 In the same experiment, screws with varying pitch-length ratios. The authors classified the test
524 screws as short ($L/S < 1.5$), medium ($1.5 \leq L/S \leq 3$), or long ($L/S > 3$) and compared performance. On the
525 laboratory-scale, the longest screws performed the best. The authors theorise that the screw buckets
526 have more time to fully form and equilibrate, while the screw has more time to convert pressures into
527 mechanical torque when the screw is longer. However, a longer screw is heavier, more expensive, and
528 experiences more friction loss. So, in practice, real world screws tend to be “medium” length in terms of
529 the above study.

530 **4.5. Number of blades**

531 Most screw generators are installed with 3, 4, or 5 bladed screws. It was shown that theoretically
532 more blades yields higher efficiencies for pumps [77], and the same could probably be suggested for
533 generators. However, the addition of blades adds significant costs to manufacturing and installation. The
534 authors have visited one 2-bladed installation at the Cragside House in Rothbury, UK. It is an
535 exceptionally long screw ($L/S \approx 11$); having less blades would reduce overall costs. Overall, it seems that
536 many screws are installed with 3 or 4 blades, this is likely the balance between performance and cost;
537 ultimately, the goal is to produce power the most economical way.

538 As mentioned above, in follow-up discussions the builder of the Romney Weir installation reflected
539 that the powerplant would likely produce more power if it had 4 blades (and not 5).

540 A CFD study tested 3-, 4-, and 5-bladed screws and varying inclination angles. The 4- and 5-bladed
541 screws had very similar efficiencies throughout – they were notably higher efficiencies than the 3-bladed

542 screw. At some inclination angles the 5-bladed screw performed with a slightly higher efficiency; the
543 additional blade was suggested to help minimize leakage losses in the screw, improving performance.
544 The authors suggest that the optimal number of blades is dependent on-site conditions. In some cases,
545 the efficiency increase and cost of additional blades improves ROI, in other cases ROI may be improved
546 by foregoing the slight efficiency increase to save the additional costs.

547 **4.6. Multiple screw installations**

548 Some powerplants have multiple, parallel screws; depending on site conditions, two or mor smaller
549 screws may perform better, or yield a better ROI than one large screw. For example, sites with
550 proportionally low head and high flow would require a large diameter screw to support the desired flow
551 rate. But, to maintain appropriate D_i/D_o and S/D_o ratios, the screw would need to be very short (which
552 corresponds to low efficiencies). In this case, multiple screws with smaller diameters could be installed
553 to support the higher flow rate. By extension, the smaller diameter screws would be proportionally
554 longer, corresponding to higher efficiencies.

555 Sites with highly variable flow conditions and large seasonal flow variation may perform better with
556 multiple screws as well. Depending on site conditions, it may be advantageous to operate one screw at
557 the year-round baseflow and additional screws as flow rates increase. However, this can sometimes be
558 accomplished using a variable speed system.

559 **4.7. Variable speed systems**

560 Variable speed ASG installations are very common, it allows the screw to accommodate changing
561 flow and head conditions at the site. Variable speed systems use electronic controls to vary the speed of
562 operation such that the screw operates at the optimal point for the given flow and head conditions. As
563 such, it is more difficult to match grid frequencies. At a high level, the power system has a raw
564 alternating current (AC) power produced by the screw that is not likely to be operating at the grid

565 frequency. The power is then passed through a rectifier, converting it to direct current (DC) and finally
566 passed through an inverter to convert it to AC power that matches the grid frequency. This process has
567 been observed to add electrical power losses up to approximately 10% [78]. Some ASG powerplants
568 have system overrides to improve plant performance; they bypass the inverter-rectifier loop when the
569 screw is operating at its design speed that matches the grid frequency.

570 Conversely, fixed-speed screws commonly use induction generators for mechanical-electrical
571 conversion. Induction generators always operate at grid frequency if the rotation speed is higher than
572 the synchronous speed. Induction generators are common for fixed-speed screws since they have a
573 lower cost of operation and are highly efficient.

574 **5. Conclusion**

575 In the end, this measurement campaign yielded a successful data point for the Ruswarp installation
576 (and two other highly uncertain data points), 6 successful data points for the Buckfast Abbey
577 installation, and some interesting and important insight on the day-to-day operations of an ASG power
578 plant. It was determined that the Ruswarp and Buckfast hydropower schemes operated (on the day of
579 measurement) with river-to-wire efficiencies of about 74.5% and 75.3%, respectively. The measured
580 efficiencies seem to be in line with the efficiency ranges given in the literature [3].

581 This campaign also provided some insight into the effect of outlet water level on screw
582 performance; though that dataset was not complete enough for model evaluation and development
583 purposes. These field studies also provided valuable qualitative information to the research team, both
584 due to assuming the role of plant operator while collecting data, and the opportunities for discussion of
585 a range of aspects of plant performance and other factors in plant operation with the operators of the
586 actual plants. Important new avenues of research were identified that were not apparent “from a
587 distance” in the literature. For example, the research team is now examining the degree of power loss

588 due to increased surface roughness on screw surfaces caused by algae growth and other contamination,
589 after this phenomenon was mentioned by two different operators at different plants.

590 This field study methodology presented in this paper would also be feasible in developing regions.
591 There is an inherent value in evaluating the performance of previously installed energy facilities: both in
592 terms of the qualitative and quantitative insights gained. For example, some of the qualitative
593 information (reported by operators) discussed in this article provided important but previously
594 unrecognised research questions, like the impact of algae on power production.

595 One of the major successes of this measurement campaign was that the measured data was able to
596 demonstrate the accuracy of a CFD model when used in a brief verification study. Altogether, this CFD
597 model may be used to extend the known dataset for full-scale installations since it was a quite involved
598 process to gather data from full-scale installations. As evidenced by this report, it took several weeks
599 worth of work (in surveying, travel, measuring, documenting, and post processing with two people),
600 access to specialized measurement equipment and major expenses (in accommodations, flights, and
601 ground transport) to yield seven reasonably accurate full-scale ASG data points.

602 This study also was able to highlight the practical design of ASG powerplants by demonstrating the
603 most common geometry and parameters for installations. Precedent and proof-of-concept are
604 invaluable for designers and may help further advance design optimization of Archimedes screw
605 generators. The successes demonstrated by the Ruswarp Hydro, Buckfast Abbey, and Totnes Hydro
606 installations indicate that Archimedes screw generators are an eco-friendly, cost-efficient, micro-
607 hydropower option. Well-designed Archimedes screw generator installations may help to grow
608 sustainable energy networks world-wide.

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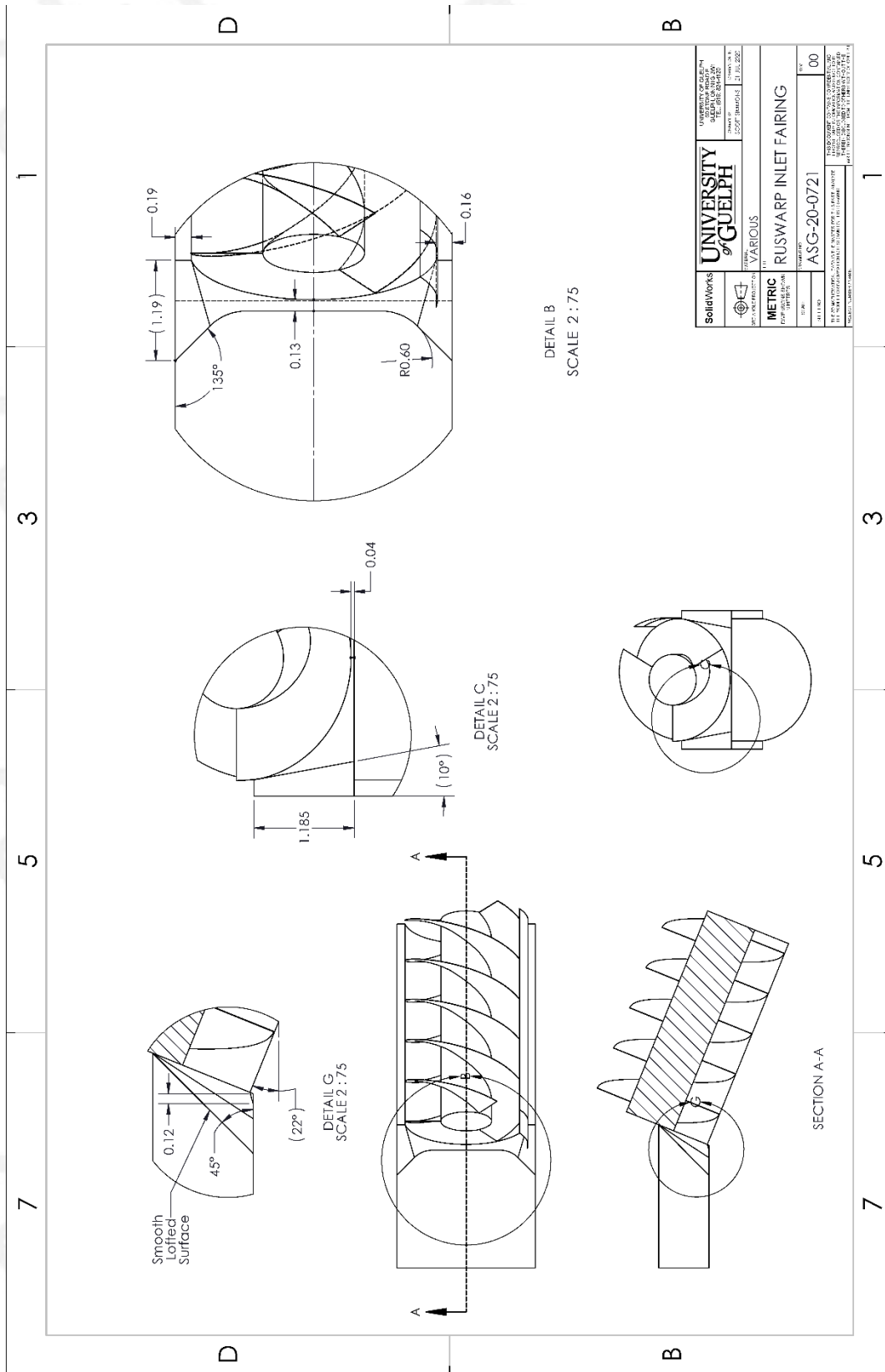
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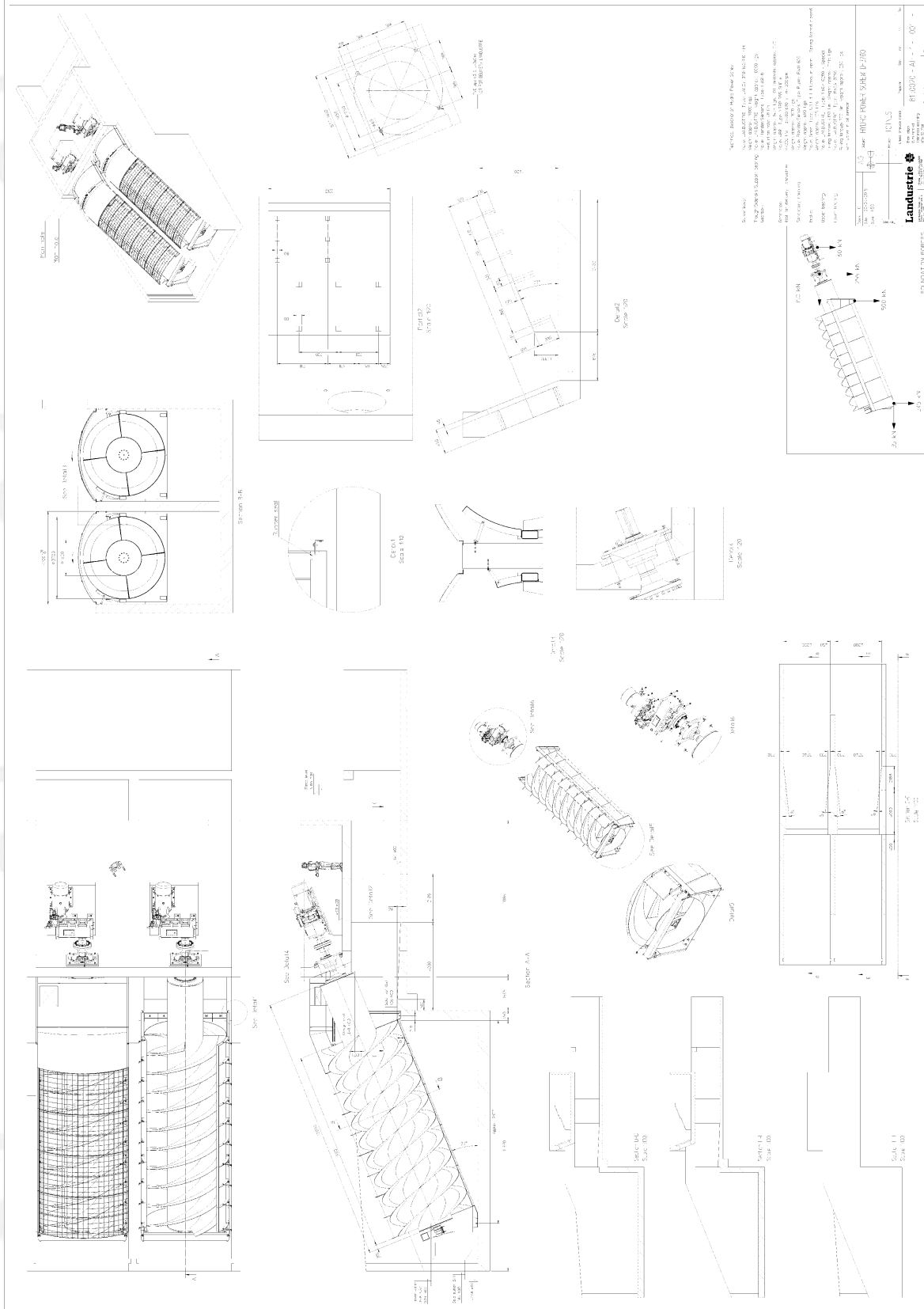
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808 Figure A 1: drawing of Ruswarp Hydro installation



809 Figure A 2: Ruswarp Hydro forebay measurements.
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815 **Figure A 4: drawings of the Totnes installation.**